# Numerical and Experimental Estimation of Measurement Uncertainty in Magnetic Field Mapping 

Paola La Marca ${ }^{\odot}$, Marco Tarabini ${ }^{\odot}$, Senior Member, IEEE, Philippe Lerch ${ }^{\odot}$, Alexander Gabard ${ }^{\circledR}$, Giuseppe Montenero ${ }^{( }$, and Ciro Calzolaio ${ }^{\left({ }^{( }\right)}$


#### Abstract

This article proposes a general method to model and simulate the process of magnetic field mapping; aim of our work is the combination of the effect of the bench position uncertainty with the magnetic measurements' uncertainty. The method is based on the study of positioner kinematics using a multi-body system approach. The geometrical errors of the manipulator, including the manufacturing tolerances and the assembling nonidealities are included in the model using the homogeneous transformation matrix (HTM), to numerically estimate the end-effector positioning uncertainty $\boldsymbol{U}_{\boldsymbol{p}}$. The positioning uncertainty is then combined with the magnetic measurement uncertainty using the magnetic field gradient as a sensitivity coefficient; in presence of strong field nonlinearities, the combination can be performed using Monte Carlo (MC) simulations in order to estimate how $U_{p}$ propagates to the magnetic measurement uncertainty $U_{f}$ at different positions. The method has been validated in the specific case of the compact field mapper (CFM), a bench featuring a Cartesian robot and a triaxial Hall sensor used to measure flux density maps in the accessible region of interest of the magnets of the Swiss Light Source, with an uncertainty below $0.5 \%$. The method allowed us to define, during the design phase, the characteristics of the positioning system (i.e., the mechanical positioning uncertainty $U_{p}$ ) in order to obtain the desired magnetic measurement uncertainty $\boldsymbol{U}_{f}$. Simulations and experiments in the case of a reference quadrupole are presented and discussed.


Index Terms-Accelerator magnets, measurement uncertainty, particle accelerators, position measurements.

Nomenclature

| $x, y, z[\mathrm{~mm}]$ | Positioner coordinates. |
| :--- | :--- |
| $E_{p}[\mu \mathrm{~m}]$ | Positioning error. |
| $U_{p}[\mathrm{~m} \mu \mathrm{~m}]$ | Positioning uncertainty. |
| $B_{i}[\mathrm{~T}]$ | Magnetic flux density. |
| $B_{n}, A_{n}[\mathrm{~T}]$ | Normal and skew multipoles. |
| $E_{f}[$ Magnetic units [28]] | Magnetic measurement error. |
| $U_{f}[$ Magnetic units $]$ | Magnetic measurement <br> uncertainty. |
| $S_{P}[1]$ | Magnetic sensor instrumental <br> uncertainty. |
| $S_{H P}[1]$ | Hall probe instrumental <br> uncertainty. |

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## I. Introduction

THE quality of nonhomogeneous field profiles generated by particle accelerator magnets can be assessed with point-like measurements in the region of interest [1]. Miniaturized 3-D magnetic sensors are supported by lightweight mechanical structures and afterward moved to obtain 3-D maps of the magnetic field. The relative contributions of the magnetic sensor and of the positioning system to the overall measurement uncertainty have never been studied in the scientific literature.
To date, the effects of the mechanical performances of the scanning system on the magnetic measurement are often neglected and not fully investigated. The commonly adopted approach consists of the initial selection of a 3-D magnetic sensor with the desired accuracy, followed by the design of a scanning robot with a positioning accuracy in the order of $1 \mu \mathrm{~m}$ [2], [3], [4]. The positioning errors are then experimentally characterized during the commissioning procedure, for example, [5], [6]. In the specific case of particle accelerator magnets, the measurement bench consists of miniaturized 3-D probes supported by thin and lightweight mechanical structures, designed to keep the motors and position sensors outside the magnetic field. The deflection of the probe holder due to the gravity, the probe vibration under external stimuli, and the positioning accuracy can be minimized only by designing bulky mechanical structures. In addition, the small magnet aperture requires selecting miniaturized 3-D Hall probes, whose uncertainty is larger than the one of larger magnetic sensors.

In an uncertainty-driven design, the starting point is the definition of the desired magnetic field uncertainty $U_{f}$. By knowing the mathematical model which describes how $U_{f}$ depends on the magnetic sensor uncertainty $U_{s}$ and the positioning system uncertainty $U_{p}$ it is possible to determine the magnetic and positioning uncertainties allowing to reach the desired performances. Measurement uncertainties can be determined starting using the ISO GUM [7], [8] or with more complex numerical methods during the design phase [9].

The uncertainty of the positioning system has been largely studied in the case of machining centers, in order to quantify the effects of geometric errors, thermal drifts, and cutting forces during manufacturing [10]. The common approach in this field is the development of a model describing the bias error components and the definition of a bias error compensation strategy [11]. The kinematics model can be built
using different strategies: Xiang and Altintas [12] proposed the use of the screw theory to model a five-axis machine and simplify the inverse kinematics equation; Fu et al. [13] used the differential motion matrix to obtain the influences of each axis on the tool positioning performances; exponential models are used by Fu et al. [14] to avoid singularity problems in complex robot chains. Finally, a general method consists in modeling the machine as a multibody system (MBS) composed of a series of rigid bodies connected by prismatic or rotational joints [15], [16]. Then, to describe the location of the end-effector in a usable frame, a homogeneous transformation matrix (HTM) is defined between each adjacent joint pair taking into account their relative position and orientation. Once the bench kinematic model is obtained, it can be used to simulate and forecast the final volumetric error [17].

Magnetic sensors are based on the transduction of different physical phenomena, for instance, the electromagnetic induction, Hall effect, tunnel magnetoresistance (TMR), giant magnetoresistance (GMR), anisotropic magnetoresistance (AMR), and giant magnetoimpedance (GMI), [18]. The knowledge of the three Cartesian components of the magnetic field at the same time with a single miniaturized transducer is a great challenge for the semiconductor industry, [19], [20], [21]. In literature, detection techniques such as AMR or GMI are preferred for high-frequency magnetic fields, due to their larger bandwidth, and for low magnetic fields (less than 1 mT ). For static fields, Hall probes are preferred since they show better resolution (especially for fields above 1 mT ) but shorter bandwidth. Finally, Hall probes are a good compromise between uncertainty and cost compared to the other technologies mentioned above, which are generally more expensive.

As already mentioned, to date, magnetic and positioning uncertainties have always been considered independently. This article proposes a general method to identify and quantify the impact of the mechanical system performances on the magnetic field measurement uncertainty by coupling the bench kinematic model with the magnetic field one and performing Monte Carlo (MC) simulations to propagate positioning error distributions to $U_{f}$. The scope of this analysis is the development of a comprehensive measurement model that allows to forecast final measurement uncertainty, including the mechanical structure in early considerations about the system behavior and using this study to guide the design choices. With this approach, it is possible to optimize the final positioning uncertainty to concur to the error budget with a nondominant weight with respect to the magnetic sensor contribution, thus optimizing the cost of the structure without investing in precise components that are not entailing uncertainty benefits.

The method has been developed for the upgrade of the storage ring of the Swiss Light Source (SLS-2) at the Paul Scherrer Institute (PSI), which requires the characterization of 800 magnets before their installation. The SLS-2 upgrade aims at the reduction of the beam emittance from 5000 pm to 137 pm keeping the same storage ring foot-print (i.e., same building) and undulators locations [22]. With these requirements, a new set of high-gradient magnets based on permanent blocks, normal-conducting coils, and super-conducting ones


Fig. 1. Contributions to the total magnetic measurement uncertainty.
is in the design phase. Among them, a superconducting dipole providing a longitudinal gradient has been designed, reaching a peak field of 6 T [23], [24]. The performances of these magnets have to be assessed before installation in the machine. Thus, a new magnet survey stand is currently in use for the thermal, mechanical and magnetic characterization of superconducting magnets in operating conditions [25]. Due to the need of inspecting nonhomogeneous field profiles, it is necessary to measure maps of all three components of the magnetic induction vector with a relative uncertainty $U_{f}$ of $0.1 \%-0.5 \%$ of the maximum generated field, corresponding to $10-50$ units ( 1 unit being $10^{-4}$ of reference field) [26].

## II. Proposed Method

The first step necessary to estimate the mapping uncertainty is the definition of a measurement model; for this purpose, the uncertainty sources affecting the final magnetic measurement process performances are analyzed. As represented schematically in Fig. 1, two main elements are present: magnetic sensor performances $U_{s}$ and positioning system ones, identified as $U_{p}$. The first contribution affects directly the final process yield in the form of nonlinearities of the sensor, the presence of noise on the voltage output or noncompensated bias errors due to disturbances. On the other hand, the positioning uncertainty weight depends on the local field gradient, thus it is necessary to consider the field distribution to know the effect of $U_{p}$ on $U_{f}$.

The uncertainty analysis is split into two parts: first, only the mechanical system performances are analyzed and modeled, then they are combined with the magnetic field distribution in the region to be scanned.

## A. Mechanical Effects

Six degrees of freedom are necessary to describe the position of a rigid body in space. Considering a general linear axis, when the carriage moves, its position will be affected by three linear errors and three angular ones. As shown in Fig. 2, for $Z$-axis, $\delta_{z}(z)$ describes the positioning error along the motion direction, $\delta_{x}(z)$ and $\delta_{y}(z)$ are the straightness errors along the other two directions, while $\varepsilon_{z}(z), \varepsilon_{y}(z)$, and $\varepsilon_{x}(z)$ are, respectively, the roll, yaw, and pitch angular errors. Considering a three-axis machine, each axis is described by six errors and, additionally, three more parameters are required to include the orthogonality error between each axis pair. Thus, 21 geometric parameters are required to model the machine. Considering a generic point $\overrightarrow{P_{1}}$ in space, which coordinates are expressed in the local frame 1 , it can be referred to the global


Fig. 2. Linear and angular errors for a linear axis moving in $Z$-direction.


Fig. 3. Example of open kinematic chain with global frame identified as 0 and local frames placed on each consecutive body.
frame knowing the relative orientation and location of the two reference systems. This transformation can be formulated as in the following:

$$
\begin{equation*}
\underline{\overrightarrow{P_{0}}}=\overrightarrow{T_{0,1}}+R_{0,1} \underline{\vec{P}_{1}} \tag{1}
\end{equation*}
$$

where $\underline{\overrightarrow{P_{1}}}$ and $\underline{P_{0}}$ are, respectively, the expression of point $\underline{\vec{P}}=[x, y, z]$ in local and global frames, $\overrightarrow{T_{0,1}}$ is the location vector containing the coordinates of the local frame origin with respect to the global one and $R_{0,1}$ is the 3-D rotation matrix to identify their relative orientation. This transformation can be put in matrix form, as in (2), to obtain the HTM of the first pair, named $M_{0,1}$

$$
M_{0,1}=\left[\begin{array}{ccc|c} 
& R_{0,1} & T_{0,1}  \tag{2}\\
\hline 0 & 0 & 0 & 1 \\
{[-4 e x]} & & &
\end{array}\right] .
$$

Equation (1) can then be rearranged in a more compact form as

$$
\begin{equation*}
\overrightarrow{P_{0}}=M_{0,1} \overrightarrow{P_{1}} \tag{3}
\end{equation*}
$$

where $\vec{P}_{i}=[x, y, z, 1]$ are the homogeneous coordinates of the point. For the sake of clarity, the 3-D rotation matrix is obtained by successive multiplication of $z y x$ rotations of $\psi$, $\theta$, and $\phi$ angles, resulting in the following matrix (where $c$ and $s$ are short for cos and $\sin$ ) (4), as shown at the bottom of the next page.

In the case of more complex kinematic chains, as in the case of an MBS shown in Fig. 3, the transformation from the global reference system to the $n$ th-frame can be easily achieved by multiplying in order the consecutive HTMs

$$
\begin{equation*}
M_{0, n}=M_{0,1} M_{1,2}, \ldots, M_{n-1, n} \tag{5}
\end{equation*}
$$

For each body, it is possible to write the HTM according to (2), knowing the relative locations and orientations of each
pair. By multiplying the HTMs following the kinematic chain order, it is possible to obtain the description of the end-effector position in space with respect to the selected global frame. In the case of parallel kinematics, the same formalism can be applied by analyzing each closed kinematic loop and adding to (5) the required constraints on links dimensions.

Using these formulations, it is possible to calculate the end-effector location in all the points of the measurement volume. This error is formulated as the difference between the target position $\vec{P}_{t}$ and the actual reached one $\vec{P}_{r}$, according to the following equation for each axis:

$$
\begin{equation*}
E_{p}=\left\|\vec{P}_{t}-\vec{P}_{r}\right\| \tag{6}
\end{equation*}
$$

The actual measurement point $\vec{P}_{r}$ can be obtained using the complete HTM $M_{n, 0}$ for each point $\overrightarrow{x_{i}}$ reached by the robot end-effector in the working volume

$$
\begin{equation*}
E_{p, i}=\left\|M_{n, 0}\left(\overrightarrow{x_{i, 0}}\right) \cdot \overrightarrow{x_{i, 0}}-\overrightarrow{x_{i, n}}\right\| . \tag{7}
\end{equation*}
$$

The HTM $M_{n, 0}$ is composed of position independent errors, such as the linear and angular geometric errors shown in Fig. 2, and position-dependent ones, mainly the nonperfect orthogonality among axis which leads to deviations linearly dependent on the travel length. For this reason, $M_{n, 0}$ in (7) is a function of the target point coordinates. Furthermore, it is possible to indicate with $M_{n, 0}^{*}$ the theoretical transformation from frame $n$ to the global reference 0 ; this matrix contains only the rigid offsets between the two frames, thus no error is taken into account. Equation (7) can be rewritten in global coordinates as

$$
\begin{equation*}
E_{p, i}=\left\|M_{n, 0}\left(\overrightarrow{x_{i, 0}}\right) \cdot \overrightarrow{x_{i, 0}}-M_{n, 0}^{*} \overrightarrow{x_{i, 0}}\right\| \tag{8}
\end{equation*}
$$

where $\overrightarrow{x_{i, 0}}=M_{0, n}^{*} \overrightarrow{x_{i, n}}$. In addition to the geometrical errors, vibrations play an important role in the definition of the positioning uncertainty of the end-effector. The oscillation amplitude $\partial \overrightarrow{x_{i}}$ is a time-dependent function and it will depend on the stimulated mode and on the system dynamic characteristics. Finally, errors in the position reading can be caused by the motion controller: in case the reading is time-delayed with respect to the real state of the system, an additional positioning error is present. This error is directly proportional to the scanning speed and it can be seen as the space traveled by the system while waiting for the position reading. Assuming constant velocity, $\Delta \vec{x}$ can be computed as

$$
\begin{equation*}
\Delta \overrightarrow{x_{i}}=v_{s} \cdot \tau \tag{9}
\end{equation*}
$$

where $v_{s}$ is the system speed and $\tau$ the reading delay in time and it can be included in 7 together with the oscillation error as

$$
\begin{equation*}
E_{p, i}=\left\|M_{n, 0}\left(\overrightarrow{x_{i, 0}}\right) \cdot\left(\overrightarrow{x_{i, 0}}+\partial \overrightarrow{x_{i, 0}}+\Delta \overrightarrow{x_{i, 0}}\right)-M_{n, 0}^{*} \overrightarrow{x_{i, 0}}\right\| . \tag{10}
\end{equation*}
$$

Equation (10) represents the positioning error model that includes the kinematic, dynamic, and motion controller sources.

## B. Magnetic Effects

A model of the magnetic field in the robot working volume has been developed to investigate the mechanical positioning effect onto magnetic field measurement error. In domains free of currents or magnetized materials, as the inside of the magnet aperture and sufficiently away from the coil ends, the field can be represented with its 2-D multipole expansion according to the following equation in complex form [27], [28]:
$\mathbf{B}(\mathbf{w})=\mathbf{B}(x, y)=B_{y}+i B_{x}=\sum_{n=1}^{N}\left(B_{n}+i A_{n}\right)\left(\frac{x+i y}{R_{\text {Ref }}}\right)^{n-1}$
where $\mathbf{w}=x+i y, B_{x}$, and $B_{y}$ are the induction vector Cartesian components, $B_{n}$ and $A_{n}$ are the normal and skew multipoles, $R_{\text {ref }}$ is the reference radius at which the harmonics are evaluated (or measured) and $N$ is the number of discrete components considered in the field representation. The expansion coefficients, the field multipoles, represent the harmonic components of the field which contribute to the field shape and they can be normal or skew, meaning real or imaginary contribution for each $n$.

Sensor performances (regardless of the sensing technology) are included in the model using the factor $S_{P}$ accounting for the noise of the transducer and all the bias errors not compensated and included in the uncertainty budget. In the case of a generic probe, $S_{P}$ can be computed by combining, according to the ISO GUM indications, the random error components and the systematic ones that cannot be compensated.

Concerning the spatial contribution in (11), it is then affected by the robot positioning errors, thus next equation shows the full coupled measurement model

$$
\begin{equation*}
\mathbf{B}(\mathbf{w})=S_{P} \sum_{n=1}^{N}\left(B_{n}+i A_{n}\right)\left(\frac{M_{P, 0}(\mathbf{w}) \cdot \overline{\mathbf{w}}}{R_{\mathrm{Ref}}}\right)^{n-1} \tag{12}
\end{equation*}
$$

where $M_{P, 0}(\mathbf{w}) \cdot \overline{\mathbf{w}}$ is the actual probe position. The spatial error contributions are explicitly indicated in
$\mathbf{B}(\mathbf{w})=S_{P} \sum_{n=1}^{N}\left(B_{n}+i A_{n}\right)\left(\frac{M_{P, 0}(\mathbf{w}) \cdot(\mathbf{w}+\partial \mathbf{w}+\Delta \mathbf{w})}{R_{\operatorname{Ref}}}\right)^{n-1}$
where $\mathbf{w}$ are the nominal sampling points, $\partial \mathbf{w}$ is the vibration effect and $\Delta \mathbf{w}$ is the time delay induced reading error. These coordinates are referred to as the global reference system as described in Section II-A. The magnetic measurement error is formulated as deviation, in units, of the measured field component from the nominal one obtained from the multipoles expansion in (11)

$$
\begin{equation*}
E_{f, i}=\frac{B_{i, \text { nom }}-B_{i, \text { meas }}}{B_{i, \text { nom }}} \tag{14}
\end{equation*}
$$

where $B_{i, \text { nom }}$ is the nominal field component in the target point and $B_{i, \text { meas }}$ is the measured field component in the actual probing location.

## C. Coupled Model

Following the formalism in [7], a measurand $\eta$ can be estimated by the central value of the distribution that can be attributed to it according to the measurement equation

$$
\begin{equation*}
\eta=g\left(\xi_{1}, \xi_{1}, \ldots, \xi_{k}\right) \tag{15}
\end{equation*}
$$

where $\xi_{k}$ are the input quantities influencing the output $\eta$ value. The function $g$ defines the relationship between inputs and output and it is used to propagate the input quantities probability density functions to the measurand, with the final aim to obtain the standard uncertainty of $\eta$. The law of propagation of uncertainty is applied to outputs that are characterized by a Gaussian or t-distribution, for which linear dependence exists between the influence quantities and the measurand. Even in cases where the $g$ function is linearized (i.e., by means of Taylor series approximation), it is important that the central limit theorem can be applied otherwise the measurement uncertainty defined confidence intervals are not representative. The supplement 1 of the ISO GUM, [8], suggests the usage of the MC method to overcome the limitations imposed by the law of uncertainty propagation: with this technique the probability distributions of the influence quantities are combined by numerical simulations generating random variables according to specified distributions. By random sampling the PDFs and knowing $g$, it is possible to obtain numerically the measurand PDF; normally, to have sufficient samples to get a valid estimate and standard deviation for $\eta$, the simulation requires more than $10^{5}$ runs. The MC method can be used to forecast the uncertainty of the magnetic scanning system using (7): according to the indications in [8], a PDF is assigned to each input quantity to simulate by high-rate sampling the achieved end-effector position PDF and compare it with nominal values. Input quantities for the kinematic model are the spatial coordinates and the geometrical errors of each body composing the motion chain; their PDFs are combined to get the estimate of the positioning error and its standard uncertainty. Considering the coupled magnetic-mechanical model built in the previous section and represented in (12), the measurand $\mathbf{B}$ is depending on the following influence quantities: Hall probe characteristic factor, spatial coordinates, geometrical errors effect through the kinematic chain, bench dynamics and measurement equipment. From this result and knowing the nominal field, it is possible to obtain the estimate and the standard uncertainty of the measurement error described in 14 .

## III. Case Study

The proposed method for the identification of the combined uncertainty requires the knowledge of the magnetic field to

$$
R_{n-1, n}=\left[\begin{array}{ccc}
c(\psi) c(\theta) & c(\psi) s(\phi) s(\theta)-c(\phi) s(\psi) & s(\phi) s(\psi)+c(\phi) c(\psi) s(\theta)  \tag{4}\\
c(\theta) s(\psi) & c(\phi) c(\psi)+s(\phi) s(\psi) s(\theta) & c(\phi) s(\psi) s(\theta)-c(\psi) s(\phi) \\
-s(\theta) & c(\theta) s(\phi) & c(\phi) c(\theta)
\end{array}\right]
$$



Fig. 4. Simulated magnetic measurement uncertainty map (at $95 \%$ ) for variable positioning uncertainty. The color bar shows the magnetic field uncertainty at the specific position.
be scanned; for current purposes, FEM models of the SLS-2 magnet apertures have been developed. FEM results consist of data sets comprising the mesh nodes coordinates $x, y, z$ and the corresponding field components $B_{x}, B_{y}, B_{z}$. The nodes' coordinates $x_{i, \text { nom }}$ have been perturbed with a Gaussian distribution $\mathrm{N}\left(0, U_{p}\right)$ with zero mean and uncertainty $U_{p}$ to simulate the probe positioning uncertainty

$$
\begin{equation*}
x_{i, \text { meas }}=x_{i, \text { nom }}+N\left(0, U_{p}\right) \tag{16}
\end{equation*}
$$

In this notation, $x_{i}$ is a generic coordinate $x, y$ or $z$, in the measurement or the nominal sets according to the subscripts meas and nom. The measured magnetic field is then interpolated at the position $x_{i, \text { meas }}$, and its difference from the magnetic field at the position $x_{i, \text { nom }}$ is considered as non-compensated bias error to be added to the uncertainty budget. Simulations were run for different $U_{p}$, in order to obtain the maximum uncertainty allowing to respect the uncertainty budget. $U_{p}$ has been varied between 0 and $50 \mu \mathrm{~m}$ with a step size of $10 \mu \mathrm{~m}$; results of simulations are shown in Fig. 4, that represents the extended uncertainty (with $95 \%$ confidence interval) in the magnet longitudinal plane, where the field exhibits the strongest gradient.

Positioning uncertainties of 20 and $50 \mu \mathrm{~m}$ allow achieving magnetic measurement uncertainties $U_{f}$ of 10 and 25 units, respectively. The final uncertainty will be determined as a combination with the uncertainty of the magnetic sensor, discussed in the next paragraphs.
The further step was the design of the scanning system [hereinafter referred to as compact field mapper (CFM)]. The selected geometry (Fig. 5) consists of a three-axis Cartesian robot based on three stacked linear stages; the magnetic sensor is mounted on a carbon fiber arm in correspondence of the robot end-effector.

1) Motion System: The motion of the bench end-effector can be analyzed considering it as the assembly of three rigid bodies (numbered as $1 \rightarrow Z$-axis, $2 \rightarrow X$-axis, $3 \rightarrow Y$-axis, in Fig. 6), on which local frames are placed, and using a global frame placed on the bench foundation (identified as 0 in


Fig. 5. SLS-2 measurement bench assembly with 1 -commercial linear stages, 2-rotation and tip-tilt stages, 3-slide, 4-sensor support, 5-rotary encoder, 6-motor, 7-ball screw, 8-linear guides.


Fig. 6. Global and local reference frames used to describe the mechanical system kinematic.

Fig. 6). The coordinate transformations from one local frame to the other can be expressed by means of HTM multiplication so that also mechanical errors (linear and angular) can be included. $X$ and $Y$ axis are commercial high-precision stages (Newport M-ILS150CC) while the $Z$-axis has been opportunely designed to cover a stroke of 800 mm . The assembly procedure has been step by step verified with the Leica Laser Tracker (LT): special attention is focused on the $Z$-axis rails mounting to verify their co-planarity (function on the plate grooves flatness specifications) and their parallelism (function of grooves sides orthogonality). Afterward, $X$ stage is mounted on $Z$-slide and, finally, $Y$-axis is stacked on $X$ one. Their mutual orthogonality is checked and adjusted iteratively thanks to the optical measurements.
2) Magnetic Sensor: The selected sensor is a 3-axial Hall probe, whose performances allow accomplishing the design requirement. The sensor characteristics are summarized in Table I.
3) Control Software: The stages are driven by Newport XPS-D motion controller for which a LabVIEW software is developed and its interface is shown in Fig. 7; the user can specify the scanning settings in terms of number of points along each axis, setup the communication parameters with the controller and the Hall probe acquisition device and define offsets to the display the results in magnet frame. The control and acquisition software has a queued message handler (QMH) architecture to simultaneously execute different threads exchanging data and commands with the XPS and, in parallel, to acquire and log the voltages from the Hall probe

TABLE I
SENIS 3-D S-Hall Probe With Analog Transducer Specifications [29], [30]

| Sensor dimensions | $10 \cdot 10 \cdot 1.4 \mathrm{~mm}$ |
| :--- | ---: |
| Magnetic field sensitive volume per axis | $150 \cdot 150 \cdot 1 \mu \mathrm{~m}$ |
| Mutual orthogonality of axes | 35 mrad |
| Range | $\pm 2 \mathrm{~T}$ |
| Sensitivity | $5 \mathrm{~V} / \mathrm{T}$ |
| Tolerance of sensitivity | $0.02 \%$ |
| Non-linearity | $0.05 \%$ |
| Offset (at zero field) | $\pm 1 \mathrm{mT}$ |
| Instrumental uncertainty | $0.15 \%$ |



Fig. 7. LabVIEW interface of CFM control and acquisition software.


Fig. 8. LabVIEW software block diagram: each subVI has its MHL and executes different tasks (trajectory planning, communication with the XPS, data acquisition).
using a multifunction NI DAQ card. Voltage sampling is regulated by evenly spaced triggers generated by XPS according to the mapping mesh specified by the operator. Furthermore, a trajectory manager is developed so that it is possible to select among scanning of lines, planes, or volumes. The block diagram is shown in Fig. 8: the central message handling loop (MHL) controls all the top-level threads, including the user interaction with the graphical interface and the display of measurement data; the motion control MHL instead generates trajectories and directly communicates to XPS control loop. The acquired voltage and position are logged in text files for post-processing.

## A. Numerical Analyses

After creating the robot kinematic model from (5), equations are coded in MATLAB to simulate the positioning error in the

TABLE II
Parameters Definition and Probability Distribution Function Associated in MC Simulation

| Axis | Geometric definition |  | PDF |  |
| :--- | :--- | :--- | :--- | :--- |
| X-Axis | Linear positioning error | $\delta_{x}(x)$ | $U(-1.5,1.5)$ | $\mu \mathrm{m}$ |
|  | Straightness error in Y | $\delta_{y}(x)$ | $U(-2.5,2.5)$ | $\mu \mathrm{m}$ |
|  | Straightness error in Z | $\delta_{z}(x)$ | $U(-2.5,2.5)$ | $\mu \mathrm{m}$ |
|  | Roll error | $\varepsilon_{x}(x)$ | $U(-18.5,18.5)$ | $\mu \mathrm{rad}$ |
|  | Yaw error | $\varepsilon_{y}(x)$ | $U(-10,10)$ | $\mu \mathrm{rad}$ |
|  | Pitch error | $\varepsilon_{z}(x)$ | $U(-18.5,18.5)$ | $\mu \mathrm{rad}$ |
| Y-Axis | Straightness error in X | $\delta_{x}(y)$ | $U(-2.5,2.5)$ | $\mu \mathrm{m}$ |
|  | Linear positioning error | $\delta_{y}(y)$ | $U(-1.5,1.5)$ | $\mu \mathrm{m}$ |
|  | Straightness error in Z | $\delta_{z}(y)$ | $U(-2.5,2.5)$ | $\mu \mathrm{m}$ |
|  | Pitch error | $\varepsilon_{x}(y)$ | $U(-18.5,18.5)$ | $\mu \mathrm{rad}$ |
|  | Roll error | $\varepsilon_{y}(y)$ | $U(-18.5,18.5)$ | $\mu \mathrm{rad}$ |
|  | Yaw error | $\varepsilon_{z}(y)$ | $U(-10,10)$ | $\mu \mathrm{rad}$ |
| Z-Axis | Straightness error in X | $\delta_{x}(z)$ | $N(5.5,7.8)$ | $\mu \mathrm{m}$ |
|  | Straightness error in Y | $\delta_{y}(z)$ | $N(3.2,6.1)$ | $\mu \mathrm{m}$ |
|  | Linear positioning error | $\delta_{z}(z)$ | $N(8.4,9.6)$ | $\mu \mathrm{m}$ |
|  | Pitch error | $\varepsilon_{x}(z)$ | $N(15.1,18.0)$ | $\mu \mathrm{rad}$ |
|  | Yaw error | $\varepsilon_{y}(z)$ | $N(15.3,15.4)$ | $\mu \mathrm{rad}$ |
|  | Roll error | $\varepsilon_{z}(z)$ | $N(10.2,16.5)$ | $\mu \mathrm{rad}$ |
|  | Orthogonality X and Y | $\alpha_{x y}$ | $N(32.3,12.8)$ | $\mu \mathrm{rad}$ |
|  | Orthogonality X and Z | $\alpha_{x z}$ | $N(51.4,11.0)$ | $\mu \mathrm{rad}$ |
|  | Orthogonality Y and Z | $\alpha_{y z}$ | $N(16.5,7.2)$ | $\mu \mathrm{rad}$ |

working volume: for each required point of the target scanning mesh, it is possible to obtain the real reached node coordinates, giving as input the entity of geometric errors.
Knowing the probability density function of each geometric error, it is possible to run the MC simulation to have the PDF of $E_{p}$ and extract the expected value (systematic part of $E_{p}$ ) and its standard deviation $\left(U_{p}\right)$. The MC simulation iteration number is set to $10^{6}$, sampling the geometric errors at each run from their PDFs and repeating the loop for 936 points distributed in the robot working volume. Table II shows all the influence variables with their PDF: uniform distribution is assigned to values given by the metrology reports of commercial stages, while normal distribution to the in-house performed inspections. Vibration effects $\partial \vec{x}$ are introduced in the analysis as zero-mean normal distribution with standard deviation of $2.5 \mu \mathrm{~m}$, obtained from a dynamic model of the bench; similarly, position reading delay effect $\Delta \vec{x}$ is modeled with a normal distribution, zero-mean and standard deviation of $1 \mu \mathrm{~m}$. These effects have been added to the simulation but their contribution can be considered negligible with respect to the impact of kinematic errors.

The largest positioning error was obtained along the $X$-axis, with a systematic error of $16 \mu \mathrm{~m}$ and a standard uncertainty of $13 \mu \mathrm{~m}$. The combination of these two quantities led to an uncertainty $U_{p}$ of $21 \mu \mathrm{~m}$.

The global measurement model is then studied by coupling the kinematic and the magnetic ones. Transducer performances have been derived from the manufacturer data-sheet (Table I). Temperature effects, that are generally relevant for Hall transducers, have not been considered because of the online sensitivity compensation implemented in the sensor. A similar consideration applies to the inductive effects due to planar Hall effect. $S_{P}$ was therefore computed from the data declared by the manufacturer that already included the sensor noise and the residual nonlinearities. Performances of the Hall probe, hereinafter referred to as $S_{H P}$, were simulated


Fig. 9. Quadrupole QFDp4.

TABLE III
Harmonic Content of QFDp4 Measured With Rotating Coils

| Index | $B_{n}$ | $A_{n}$ |
| :--- | ---: | ---: |
|  |  |  |
| $1(\mathrm{~T})$ | $5.80 \times 10^{-8}$ | $9.11 \times 10^{-9}$ |
| $2(\mathrm{~T})$ | $6.96 \times 10^{-2}$ | 0 |
| 3 | $2.85 \times 10^{-1}$ | 1.25 |
| 4 | $-3.76 \times 10^{-1}$ | $-1.28 \times 10^{-1}$ |
| 5 | $-7.94 \times 10^{-1}$ | $6.91 \times 10^{-1}$ |
| 6 | -5.55 | $5.61 \times 10^{-1}$ |
| 7 | $-3.29 \times 10^{-2}$ | $3.85 \times 10^{-2}$ |
| 8 | $-8.37 \times 10^{-2}$ | $-7.19 \times 10^{-2}$ |
| 9 | $-1.36 \times 10^{-1}$ | $1.70 \times 10^{-1}$ |
| 10 | $3.60 \times 10^{-1}$ | $-5.59 \times 10^{-1}$ |
| 11 | $3.62 \times 10^{-2}$ | $3.43 \times 10^{-2}$ |
| 12 | $-1.45 \times 10^{-2}$ | $-1.66 \times 10^{-2}$ |
| 13 | $-2.12 \times 10^{-2}$ | $-6.96 \times 10^{-3}$ |
| 14 | $1.96 \times 10^{-1}$ | $-1.29 \times 10^{-1}$ |
| 15 | $1.09 \times 10^{-2}$ | $1.00 \times 10^{-2}$ |

by a uniform probability distribution with a standard deviation equal to $0.15 \%$ and centered on the nominal unit value.

The MC simulation is performed also in this case with $10^{6}$ iterations and the PDF of $E_{f}$ is processed to get the expected value (systematic part of $E_{f}$ ) and its standard deviation $\left(U_{f}\right)$. Concerning the multipoles, this study is applied to a quadrupole magnet that has been fully characterized in the past at PSI for the SwissFEL project. The magnet, called QFDp4 and shown in Fig. 9, has a main quadrupole component and its first 15 harmonics are measured with the rotating coil technique [31] and listed in Table III. The simulated magnetic measurement resulted in 4 units of systematic error and a standard uncertainty of 5 units, which are combined to calculate the magnetic field measurement uncertainty $U_{f}$ as 6.5 units.

## B. Experimental Results

The volumetric error is mapped using a Leica LT to investigate the positioning performances of the CFM; The metrological performances of the Tracker are declared by the manufacturer and include a maximum error of $10 \mu \mathrm{~m}$ and a standard uncertainty of $5 \mu \mathrm{~m}$. The target retro-reflector is mounted on the bench end-effector and 936 target points are

TABLE IV
Systematic and Random Positioning Error Components Obtained in Simulations and Experiments

|  | Experiments |  |  | Simulation |  |  |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| $E_{p}[\mu \mathrm{~m}]$ | X-Axis | Y-Axis | Z-Axis | X-Axis | Y-Axis | Z-Axis |
| Systematic | 7 | 11 | 11 | 16 | 8 | 8 |
| Uncertainty | 13 | 13 | 14 | 11 | 11 | 13 |



Fig. 10. Average systematic component of CFM volumetric positioning error.
commanded to the robot in ten repetitions: these locations are spaced along each axis to cover the whole working volume. The software commands the robot to reach the target points and their actual coordinates are measured with the laser tracker using a 2 s time average. The scanning time was selected as a tradeoff between a reasonable test duration and the increase of the reference position accuracy deriving from the averaging procedure. Fig. 10 shows the average of the positioning error obtained as the difference between the nominal target position and the coordinates acquired by the laser tracker in the volume.

Table IV reports the comparison of the positioning error components (bias and random) between experimental results and simulations; in this case, the largest error was obtained along $Z$-axis, with a systematic error of $11 \mu \mathrm{~m}$ and a standard uncertainty of $14 \mu \mathrm{~m}$. The combination of these two quantities led to an uncertainty $U_{p}$ of $18 \mu \mathrm{~m}$. Experimental results are compatible with the simulation ones along all three Cartesian axes; the larger difference occurs along $X$-axis, where the systematic error component measured during the experiments is lower than that expected from simulations. This is reasonable due to an overestimation of $X$-related values reported in Table II; the identification of the reasons that led to the discrepancy between experiments and simulations could be performed using factorial design of experiments techniques and MC methods, as in [9].

The QFDp4 quadrupole is used then to test the accuracy of the CFM and to validate the measurement model. The magnet is installed in the measurement area and its orientation in space is adjusted thanks to mechanical jigs and checked with the LT. After this alignment, the orientation and location of the magnet frame are defined with respect to the Hall probe ones. Maps of the generated field in the magnet aperture are acquired with 30 repetitions and Fig. 11 shows an example


Fig. 11. Example of a typical magnetic field measured by the CFM: 2-D map of magnetic field $y$-component generated by QFDp4 on its mid-plane.

TABLE V
Systematic and Random Magnetic Error Components Obtained in Simulations and Experiments

|  | Experimental $E_{f}$ [units] | Simulated $E_{f}$ [units] |
| :--- | :---: | :---: |
| Systematic | 4 | 4 |
| Uncertainty | 6 | 5 |



Fig. 12. Comparison simulated and experimental positioning error, indicated with MC and LT respectively. The uncertainty is reported with a confidence interval of $95 \%$.
of a 2-D magnetic map. The plot shows the dependence of the magnetic field component along the $Y$-axis from the $x$ and $z$ coordinate and is a representative situation of the use of the CFM, with strong gradients along different axes. The measured field is then compared with the reference values obtained with the rotating coil system and reported in Section III-A; systematic error and experimental standard uncertainty of the magnetic measurement experiments and simulation are reported in Table V. For the experimental case, bias and random components can be combined as $U_{f}$ reaching 7.5 units.

## C. Method Validation

This section compares the results obtained in the simulation and in the experimental environments to validate the measurement model and assess the final performances of the CFM. Fig. 12 shows the systematic positioning error and its uncertainty with $95 \%$ confidence intervals obtained from simulation and experiments for each axis. The plot shows the compatibility of results, thus validating the kinematic model


Fig. 13. Comparison of simulated and experimental total measurement error for the CFM; the systematic component of $E_{f}$ is reported together with its uncertainty extended to $95 \%$ confidence interval.
of the robot. The CFM positioning uncertainty $U_{p}$ is verified to be $36 \mu \mathrm{~m}$, with a coverage factor of 2 .

Fig. 13 shows the $95 \%$ confidence intervals for the simulated and the experimental scans of the induction vector produced by QFDp4 in its aperture. Also, in this case, the intervals are overlapped showing good agreement and finally the magnetic measurement uncertainty $U_{f}$ is 15 units, with a coverage factor of 2 .
Both the kinematic model and the coupled mechanicalmagnetic ones are validated with the experimental campaign and, finally, it is proven that $U_{f}$ is well below 50 units as required for the SLS-2 upgrade.

## IV. DISCUSSION

## A. Model Validity

As presented in the previous section, the results obtained from the numerical estimation of the magnetic measurement uncertainty are coherent with the experimental ones, thus proving the method validity. With this technique, it is possible to forecast the final process yield and adjust the design parameters to achieve the target measurement performances. This result can be obtained only with a deep understanding of the positioner kinematic, that, as in the analyzed case, is simplified by a serial chain, but can show high complexity in more articulated robots. Furthermore, the magnetic model used to complete the measurement function is only bi-dimensional, neglecting the field distribution variation along the particle direction due to fringe effects caused by the yoke and the coil ends. These two limitations do not represent a violation of the method applicability but require further efforts to model the mechanical and the magnetic elements. In this work, the interferences caused by the magnetic field on the bench mechanical components are neglected due to the design of a long measurement arm. Generally, these effects should be considered by modeling and measuring the motor torque variation when put in proximity of a magnet to be tested.

## B. Recommendations

The design of a magnetic field scanning system should target the minimization of the design efforts, to avoid requirements stricter than necessary. This step requires knowledge of
the magnetic field gradients. In presence of limited gradients, or if the magnetic field is constant, the uncertainty of the positioning system is not affecting the magnetic measurement, given that a position error does not entail an error in the magnetic field magnitude. In this case, the sensor performances are driving the uncertainty budget, thus requiring less restrictive mechanical constraints on tolerancing. In the case of high gradients, the mechanical structure becomes crucial in the instrument design, since a small error in the position causes a large error in the magnetic field magnitude; rigid structures, in these situations, ensure limited static deflections and acceptable dynamic effects. Slender structures, in fact, increase the jitter generated by the mechanical vibrations induced by the probe motion. Another limit arises as a consequence of the larger static deflection, which requires complex compensation techniques and time-consuming calibration. In all the intermediate cases, the coupled mechanical-magnetic model is a fundamental tool to have a good tuning between the two contributions and fulfill the $U_{f}$ requirement.

## V. CONCLUSION

This article proposed a method to investigate the effect of mechanical positioning uncertainty on magnetic field maps. The method allows the a priori estimation of the final measurement uncertainty according to the mechanical design choices. This study has been applied to the CFM developed for SLS-2 magnets: having a target measurement uncertainty for the system under design, the weight of the position uncertainty with respect to the magnetic sensor one is opportunely tuned to avoid spoiling the sensor performances. For a stated magnetic sensor uncertainty of 15 units ( $0.15 \%$ ), the CFM propagated measurement uncertainty $U_{f}$ is within the sensor limit. Currently, the bench is installed in the magnetic measurement laboratory at PSI and it is ready for serial measurements.

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Paola La Marca was born in Naples, Italy, in April 1993. She received the master's and Ph.D. degrees (cum laude) in mechanical engineering from the Politecnico di Milano, Milan, Italy, in 2017 and 2021, respectively.
From 2017 to 2021, she was a visiting Ph.D. student at the Magnet Section, Paul Scherrer Institute, Würenlingen, Switzerland, where she completed the Ph.D. thesis on 3-D magnetic field mapping systems. Her research is focused on magnetic measurement techniques for particle accelerator magnets, especially on uncertainty driven magnetic measurement bench design. Further research interests concern metrology, automation and robotics, and motion systems design and control.


Marco Tarabini (Senior Member, IEEE) was born in Lecco, Italy, in June 1978. He received the master's degree in mechanical engineering and the Ph.D. degree (cum laude) in engineering of mechanical systems from the Politecnico di Milano, Milan, Italy, in 2003 and 2007, respectively.
He is currently an Associate Professor with the Politecnico di Milano. He has authored two book chapters, more than 50 scientific works published in peer-reviewed journals and more than 60 papers published in conference proceedings. He is the co-inventor of three patents and scientific coordinator of the two Joint Research Centers of Politecnico di Milano. His research interests include the study of measurement uncertainty in industrial applications, on the study of the response of the human body to vibration, and on the design of measurement instruments to be used in harsh environments.

Philippe Lerch was born in Neuchâtel, Switzerland, in April 1957. He received the Diploma degree in physics and the Ph.D. degree in science (condensed matter physics) from the University of Neuchâtel, Neuchâtel, Switzerland, in 1980 and 1987, respectively.

After his Post-Doctoral time spent at the University of California at Berkeley, Berkeley, CA, USA, investigating the potential of cuprate high temperature superconductors as detectors he joined the Laboratory of Astrophysics, Paul Scherrer Institute (PSI), Würenlingen, Switzerland, in 1992 to work on the application of low temperature superconducting devices as quantum photon detectors. In 2005, his interest switched to IR spectroscopy using synchrotron radiation, available at PSI. He finalized, with a team, the build of the IR beam line and lead its scientific operation until 2014 when he joined the Magnet Section of PSI to work on a magnetic measurement project for ITER, in collaboration with CERN, Geneva. In 2017, he started, with his team, investigations on the feasibility of using permanent magnet materials to build larger series of accelerator magnets required for the up-grade of the synchrotron radiation source. He has authored or coauthored some 80 scientific peer-reviewed publications.


Alexander Gabard received the bachelor's degree in mechanical engineering from the University of Applied Sciences and Arts of the Northwestern Switzerland, Windisch, Switzerland, in 1998.
He joined the Magnet Design Section, Paul Scherrer Institute, Würenlingen, Switzerland, in 2005, where he is the Leader of the Magnet Design and Measurements Group. He authored 15 scopus-indexed papers. His research interests include magnetic measurements and magnets' design.


Giuseppe Montenero was born in Caserta, Italy, in July 1979. He received the master's degree in telecommunications engineering and the Ph.D. degree (cum laude) in information engineering from the University of Sannio, Benevento, Italy, in 2008 and 2013, respectively.
His career in the field of magnets' technologies and measurements started at the European Organization for Nuclear Research (CERN), in October 2007. He joined the Magnet Section, Paul Scherrer Institute, Würenlingen, Switzerland, in May 2017, where he is currently a Magnet Engineer. His research interests include spanned from evolutionary algorithms, diagnostics for digital instrumentation, and software to the characterization of superconducting cables and magnets. The focus of his research work concerns now high accuracy measurement systems for accelerator magnets. In his areas of expertise, he has authored or coauthored 16 scientific works published in peer-reviewed journals and several papers published in conference proceedings.


Ciro Calzolaio was born in Cisternino, Italy, in December 1984. He received the master's degree in energy and nuclear engineering from the University of Bologna, Bologna, Italy, in 2009, and the Ph.D. degree in physics from École Polytechnique Fédérale de Lausanne (EPFL), Lausanne, Switzerland, in 2013.
He joined the Magnet Section, Paul Scherrer Institute, Würenlingen, Switzerland, in 2005, where he has been with the Measurements and Analysis Group since 2020. His research interests include the design and construction of resistive, permanent magnets, and superconducting magnets. In his area of interest, he is the author or coauthor of more than 20 papers published in peer-reviewed journals or conference proceedings.


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    Paola La Marca and Marco Tarabini are with the Department of Mechanical Engineering, Politecnico di Milano, 20156 Milan, Italy (e-mail: marco.tarabini@polimi.it).
    Philippe Lerch , Alexander Gabard, Giuseppe Montenero, and Ciro Calzolaio are with the Department of Magnet Design and Measurements, Paul Scherrer Institute, 5232 Würenlingen, Switzerland.

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